



Palestine Technical University- Kadoorie (PTUK)

Mechanical Engineering Department

12210244: Dynamics

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This is an explanation of the Dynamics course
offered at Palestine Technical University - Kadoorie

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Textbook:

Engineering Mechanics: Dynamics, 7th Edition

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Chapter Six: Plane Kinetics of Rigid Bodies

Section Six: Work-Energy Relations

6.6 Work-Energy Relations

- Work

$$U_{1 \rightarrow 2} = \int_1^2 \vec{F} \cdot d\vec{r}$$

$$U_{1 \rightarrow 2} = \int_{\theta_1}^{\theta_2} M d\theta$$

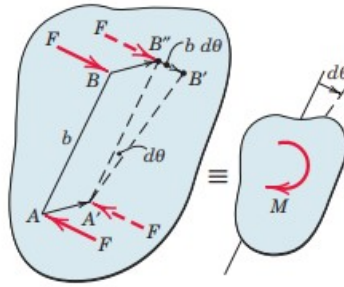


Figure 6/11

- Kinetic Energy

- Translation

$$T = \frac{1}{2}mv^2$$

- Fixed-axis rotation

$$T = \frac{1}{2}I_O\omega^2$$

- General plane motion

$$T = \frac{1}{2}mv_G^2 + \frac{1}{2}I_G\omega^2 = \frac{1}{2}I_C\omega^2$$

- Potential Energy and the Work-Energy Equation

$$T_1 + U_{1 \rightarrow 2} = T_2$$

$$T_1 + V_1 + U'_{1 \rightarrow 2} = T_2 + V_2$$

- Power

$$P = \vec{F} \cdot \vec{v}$$

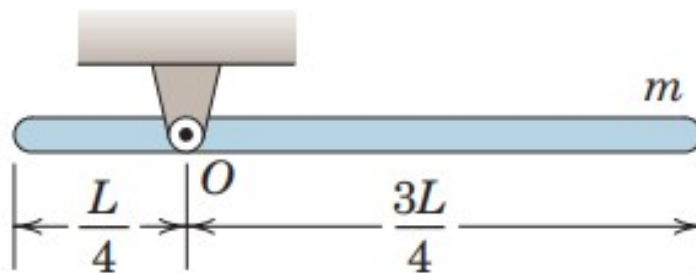
$$P = M\omega$$

End of Section 6.6

Example 1:

The uniform slender bar of mass $m = 5 \text{ kg}$ and length $L = 4 \text{ m}$ is released from rest when in the horizontal position shown. Determine its angular velocity and mass center speed as it passes the vertical position.

ans. $\omega = 2.9 \text{ rad/sec CW}$ $v_G = 2.9 \text{ m/sec}$



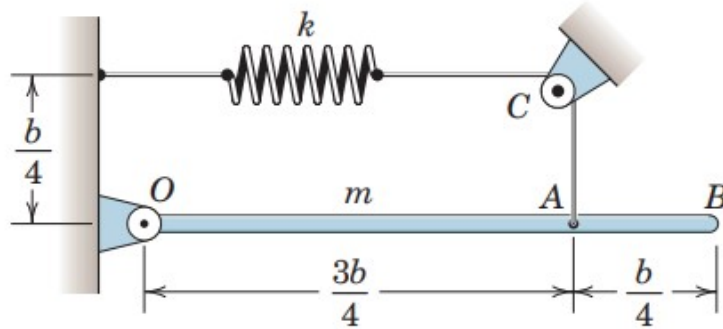
Ans.

Ans.

Example 3:

The uniform slender bar of mass m pivots freely about a horizontal axis through O . If the bar is released from rest in the horizontal position shown where the spring is unstretched, it is observed to rotate a maximum of 30° clockwise. The spring constant $k = 200 M/m$ and the distance $b = 200 \text{ mm}$. Determine: (a) the mass m of the bar and (b) the angular velocity of the bar when the angular displacement is 15° clockwise from the release position.

ans. $m = 1.196 \text{ kg}$ $\omega = 4.36 \text{ rad/sec CW}$



Ans.

Chapter Six: Plane Kinetics of Rigid Bodies

Section Seven: Impulse-Momentum Equations

6.7 Impulse-Momentum Equations

- Linear Momentum

$$\vec{G} = m\vec{v}_G$$

$$\sum \vec{F} = \dot{\vec{G}}$$

$$\sum F_x = \dot{G}_x \quad \sum F_y = \dot{G}_y$$

$$\vec{G}_1 + \int_{t_1}^{t_2} \sum \vec{F} dt = \vec{G}_2$$

$$(G_x)_1 + \int_{t_1}^{t_2} \sum F_x dt = (G_x)_2$$

$$(G_y)_1 + \int_{t_1}^{t_2} \sum F_y dt = (G_y)_2$$

- Angular Momentum

$$H_G = I_G\omega$$

$$\sum \vec{M}_G = \dot{H}_G$$

$$(H_G)_1 + \int_{t_1}^{t_2} \sum M_G dt = (H_G)_2$$

$$H_O = I_G\omega + mv_Gd$$

$$H_O = I_O\omega$$

$$\sum \vec{M}_O = \dot{H}_O$$

$$(H_O)_1 + \int_{t_1}^{t_2} \sum M_O dt = (H_O)_2$$

- Conservation of Momentum

$$G_1 = G_2$$

$$(H_G)_1 = (H_G)_2$$

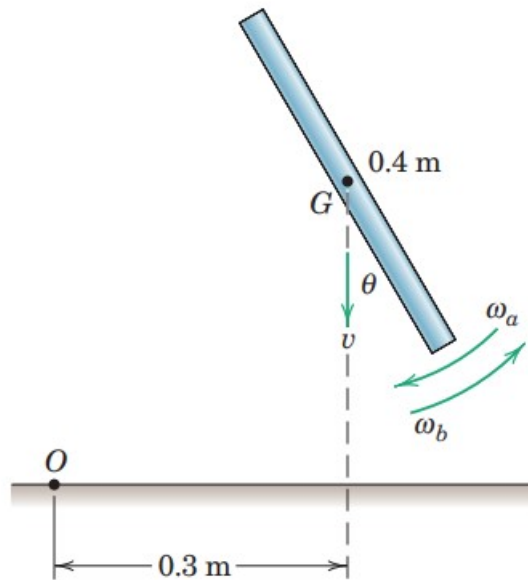
$$(H_O)_1 = (H_O)_2$$

End of Section 6.7

Example 1:

The mass center G of the slender bar of mass 0.8 kg and length 0.4 m is falling vertically with a velocity $v = 2 \text{ m/sec}$ at the instant depicted. Calculate the angular momentum of the bar about point O if the angular velocity of the bar is (a) $\omega_a = 10 \text{ rad/sec}$ clockwise and (b) $\omega_a = 10 \text{ rad/sec}$ counterclockwise

ans. a) $H_O = 0.487 \text{ Kg.m}^2/\text{sec}$ b) $H_O = 0.373 \text{ Kg.m}^2/\text{sec}$

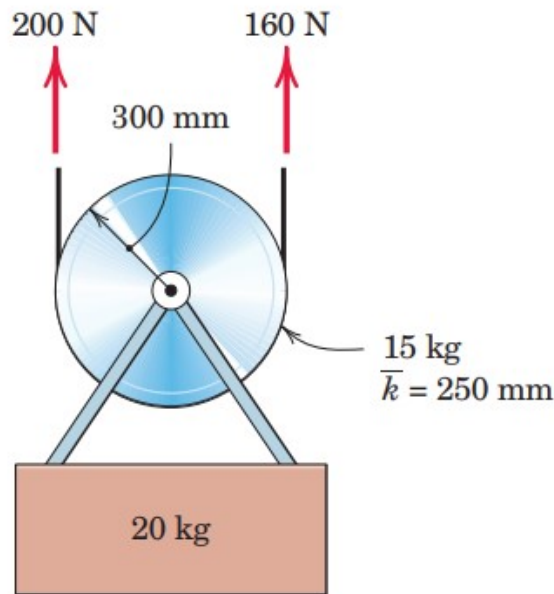


Ans.

Example 2:

The constant tensions of 200 N and 160 N are applied to the hoisting cable as shown. If the velocity v of the load is 2 m/s down and the angular velocity ω of the pulley is 8 rad/s counterclockwise at time $t = 0\text{ sec}$ determine v and ω after the cable tensions have been applied for 5 s . Note the independence of the results

ans. $v = 0.379\text{ m/sec Up}$ $\omega = 46\text{ rad/sec CW}$



Ans.

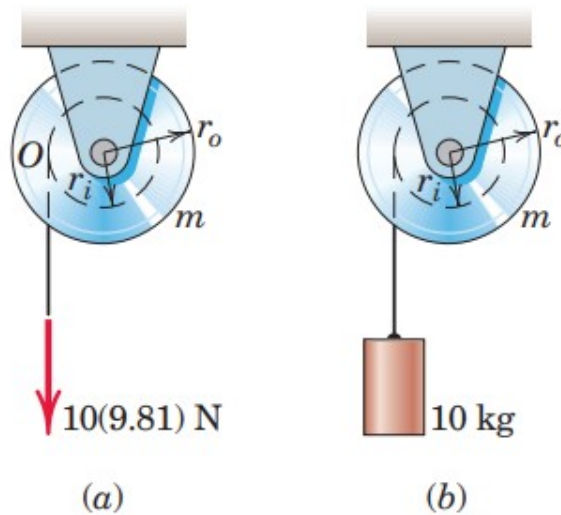
Example 3:

The grooved drums in the two systems shown are identical. In both cases, (a) and (b), the system is at rest at time $t = 0$ sec. Determine the angular velocity of each grooved drum at time $t = 4$ sec. Neglect friction at the pivot O .

ans. a) $\omega = 119 \text{ rad/sec CCW}$ b) $\omega = 72 \text{ rad/sec CCW}$

$$m = 14 \text{ kg}, \bar{k} = 225 \text{ mm}$$

$$r_o = 325 \text{ mm}, r_i = 215 \text{ mm}$$



Ans.